

# 31

## Linearized Prebuckling: Limitations

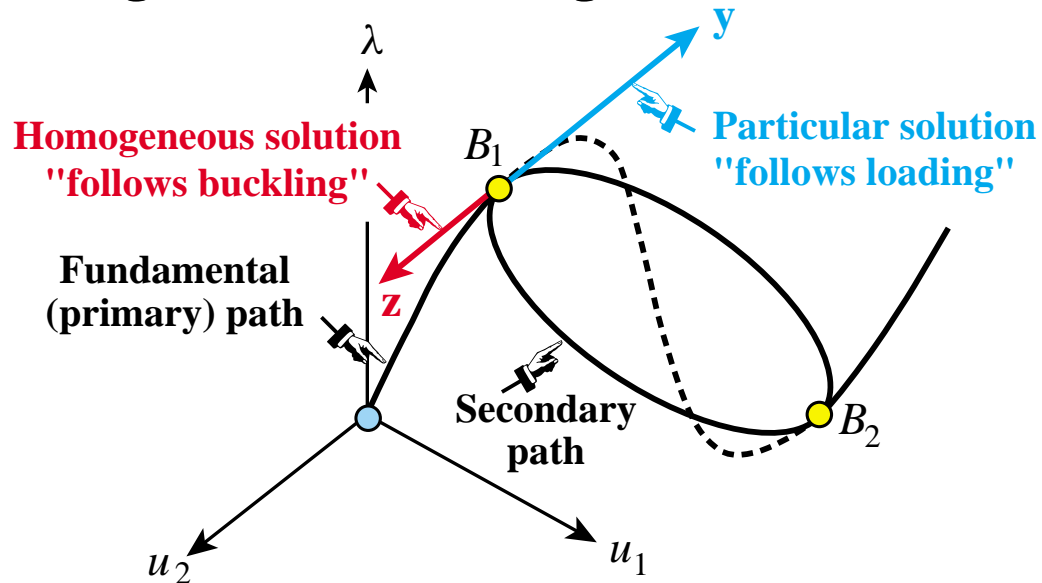
## **Bifurcation Point Analysis Levels**

**Level 0: Critical point location <- Chapter 5**

**Level 1: Tangents to emanating branches <- This Chapter**

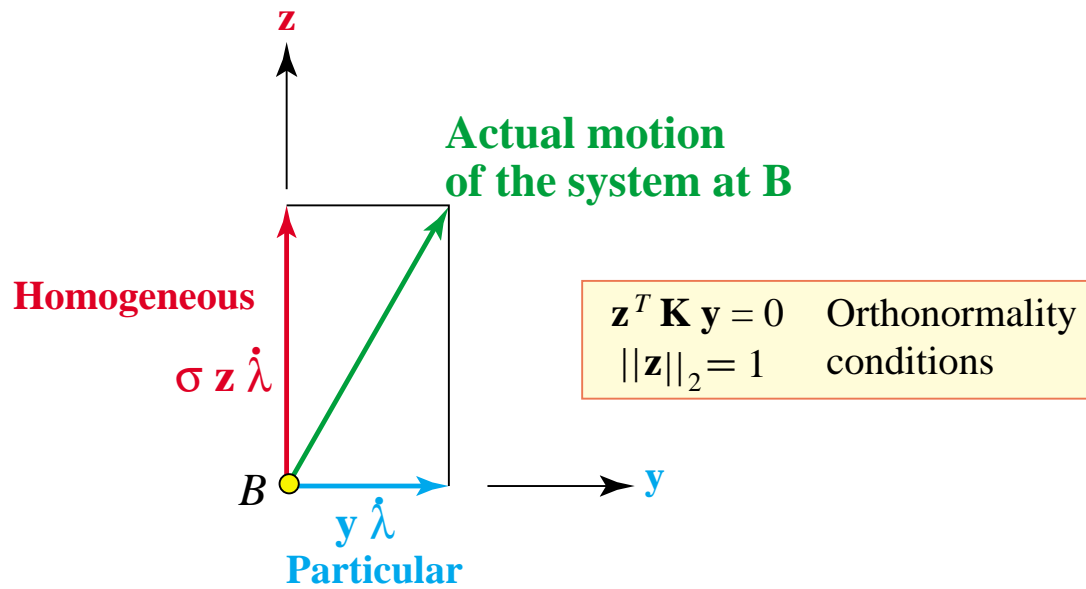
**We will only consider isolated bifurcation points**

# State Decomposition At **Isolated** Bifurcation Point: Tangents to Emanating Branches



Vectors  $\mathbf{y}$  and  $\mathbf{z}$  are mutually **orthogonal** (proved later)  
Together they define a **plane** in control-state space.  
All important physics happens in that plane, regardless  
of how many DOF the model has: 2 or a million

# Drawing the State Decomposition in the y-z Plane



# The Mathematics of Isolated Bifurcation (1)

We will assume that the first critical point found on the primary path, and characterized by the singular stiffness criterion

$$\mathbf{K}(\mathbf{u}_{cr}, \lambda_{cr}) \mathbf{z}_{cr} = \mathbf{0}$$

is an **isolated bifurcation point**.

Therefore the **normalized null eigenvector** (buckling mode)  $\mathbf{z}$ : not null,  $\|\mathbf{z}\| = 1$  is orthogonal to the incremental load vector

$$\mathbf{q}^T \mathbf{z} = \mathbf{z}^T \mathbf{q} = 0 \quad (\mathbf{z} = \mathbf{z}_{cr} \text{ for brevity})$$

Assume that we have located  $B$  and computed  $\mathbf{z}$ . Our task is to examine the behavior of the system in the vicinity (neighborhood) of  $B$ .

In this Chapter we shall be content with looking at the so-called **level-one** information: **tangents to the equilibrium branches** that emanate from  $B$

## The Mathematics of Isolated Bifurcation (2)

To carry out the task we borrow from algebraic ODE theory.  
the variation of the state vector measured from its value  $\mathbf{u}_B$   
at bifurcation is

$$\Delta \mathbf{u} = \mathbf{u} - \mathbf{u}_B$$

Divide this increment by  $\Delta t$ ,  $t$  being the pseudotime parameter,  
and pass to the limit  $t \rightarrow 0$ , where the clock starts at  $B$ :

$$\dot{\mathbf{u}} = \lim_{t \rightarrow 0} \frac{\Delta \mathbf{u}}{\Delta t}$$

## The Mathematics of Isolated Bifurcation (3)

The variation rate  $\dot{\mathbf{u}}$  from bifurcation can be decomposed into a **homogeneous solution** component  $\sigma \mathbf{z}$  in the **buckling mode direction**, and a **particular solution** component  $\mathbf{y}$ , which is orthogonal to  $\mathbf{z}$  and goes along the **incremental velocity** vector:

$$\dot{\mathbf{u}} = (\mathbf{y} + \sigma \mathbf{z}) \dot{\lambda} \quad \mathbf{y}^T \mathbf{K} \mathbf{z} = \mathbf{z}^T \mathbf{K} \mathbf{y} = 0$$

The particular solution solves the system

$$\mathbf{K} \mathbf{y} = \mathbf{q} \quad \mathbf{y}^T \mathbf{z} = 0$$

which is simply the **first-order rate equation**  $\mathbf{K} \dot{\mathbf{u}} = \mathbf{q} \dot{\lambda}$  **augmented by a normality constraint**. Imposing that constraint removes the singularity (rank deficiency) of  $\mathbf{K}$  at bifurcation point. The geometric interpretation of this decomposition on the  $\mathbf{y}, \mathbf{z}$  plane has been shown on a previous slide.

## Linearized Prebuckling Assumptions (1)

We can now restate the LPB assumptions in more precise form.

(I) The external loading is conservative and proportional:

$$\mathbf{f} = \mathbf{q}_0 + \lambda \mathbf{q}$$

while the structure is linearly elastic. This implies that the total residual equations are derivable from a potential energy function

(II) The displacements and displacement gradients prior to the critical state are negligible in the sense that (a) the **material stiffness matrix** can be evaluated in the reference configuration, and (b) the **geometric stiffness matrix** is proportional to the control parameter  $\lambda$ ,

$$\mathbf{K}_M \equiv \mathbf{K}_0 \quad \mathbf{K}_G \equiv \lambda \mathbf{K}_1$$

in which  $\mathbf{K}_1$  is the geometric stiffness for  $\lambda = 1$ , also evaluated at the reference configuration



## Linearized Prebuckling Assumptions (3)

$\mathbf{K}_0$  and  $\mathbf{K}_1$  receive the names **linear stiffness** and **reference geometric stiffness**, respectively. As discussed in the previous Chapter, the singular stiffness criterion leads to the **LPB eigenproblem**

$$(\mathbf{K}_0 + \lambda \mathbf{K}_1) \mathbf{z} = \mathbf{0}$$

(III) The **particular solution**  $\mathbf{y}$  defined previously is obtained by solving

$$(\mathbf{K}_0 + \lambda_{cr} \mathbf{K}_1) \mathbf{y} = \mathbf{q}$$

under the **orthogonality constraint**  $\mathbf{y}^T \mathbf{z} = 0$ .

Observe that from assumption (I),  $\mathbf{q}$  is constant, while from assumption (II), both  $\mathbf{K}_0$  and  $\mathbf{K}_1$  are constant. Hence the **LPB eigenproblem is linear** in the control parameter  $\lambda$

## Linearized Prebuckling Assumptions (4)

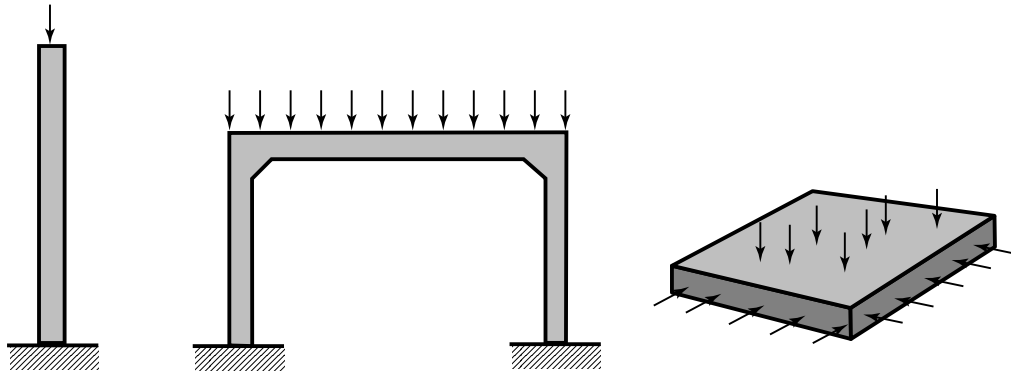
We now prove that if these assumption hold, all critical points determined from the LPB eigenproblem are bifurcation points, that is,  $\mathbf{z}^T \mathbf{q}$  vanishes. To show that, premultiply both sides of

by  $\mathbf{z}^T$

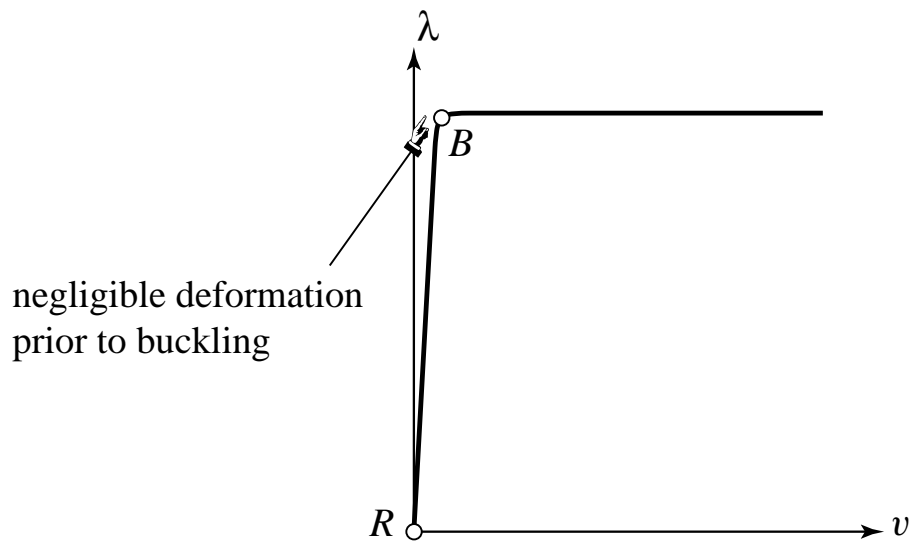
$$(\mathbf{K}_0 + \lambda_{cr} \mathbf{K}_1) \mathbf{y} = \mathbf{q}$$

$$\mathbf{z}^T \mathbf{q} = \mathbf{z}^T (\mathbf{K}_0 + \lambda \mathbf{K}_1) \mathbf{y} = \mathbf{y}^T (\mathbf{K}_0 + \lambda \mathbf{K}_1) \mathbf{z} = \mathbf{y}^T (\mathbf{K} \mathbf{z}) = 0$$

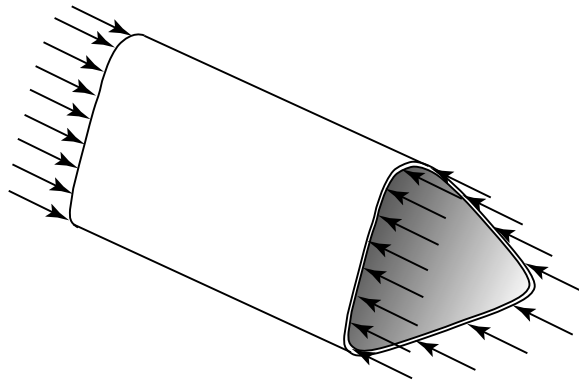
## Structures That Fit LPB Assumptions Well



## Because Of Their Kind Of Response:

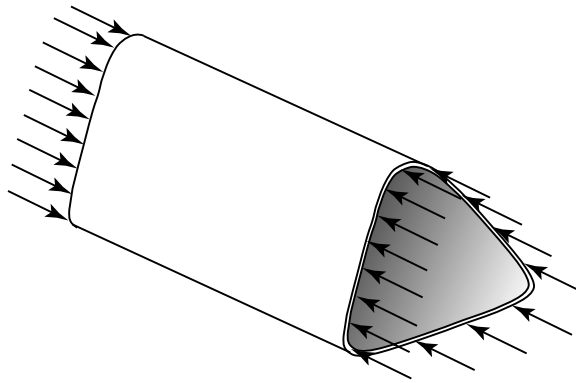


## **Some Structures That Don't Fit LPB Well (1)**

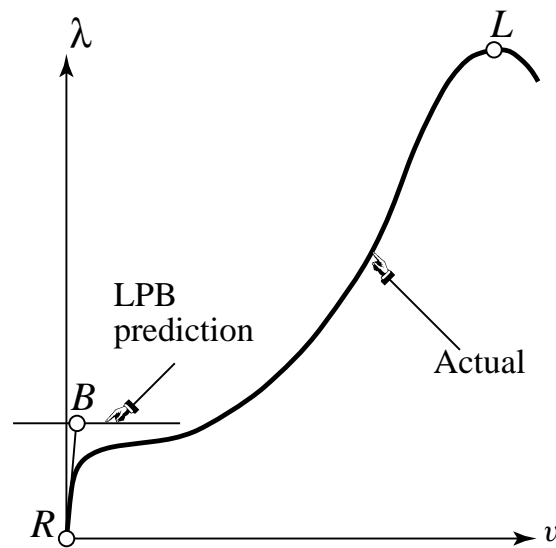


## **Some Structures That Don't Fit LPB Well**

### **Ex. 1: LPB Grossly Underestimates Strength**

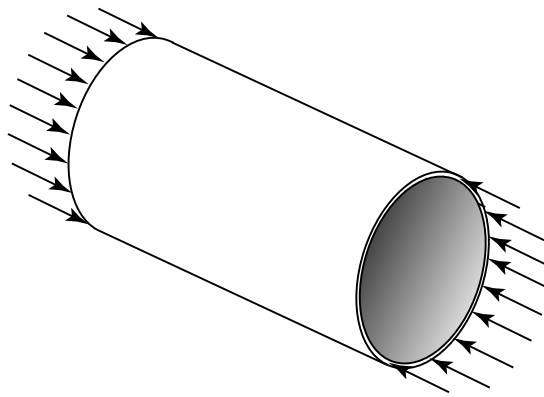


## Reason: Stress Redistribution Prior to Collapse



## **Some Structures That Don't Fit LPB Well**

### **Ex. 2: LPB Grossly Overestimates Strength**





## Reason: Extremely High Imperfection Sensitivity

